

BELLOWS TYPE MECHANICAL SEAL

CROSS-REFERNCE TO RELATED APPLICATIONS

1 This application claims the benefit of U.S.
2 Provisional Application No. 60/169,312, filed December 7,
3 1999.

BACKGROUND OF THE INVENTION

6 The present invention relates generally to sealing
7 systems for rotating machines and, more specifically, to
8 an improved sealing assembly used therein.

9 Sealing systems are often used in machine
10 applications to provide a seal between a rotating shaft
11 and a machine housing, wall or other stationary element
12 of a machine. Such machine applications include, but are
13 not limited to, fluid pumps and fluid mixing machines.
14 The seal may be provided to prevent fluids being worked
15 on, such as a liquid being pumped, from entering either
16 the drive mechanism of the machine or the atmosphere.

17 These machines generally include a stationary
18 element (such as a housing), a drive element (such as a
19 shaft), and a driven element (such as an impeller)
20 connected to the drive element. One way to provide a
21 rotatable seal between the drive element and the
22 stationary element is by providing two sealing rings and
23 leaf springs. One ring forms a seal with a sealing face
24 that is rigidly attached to the stationary element. The
25 other ring forms a seal with a sealing face that is fixed
26 to the driven part. The two sealing rings are biased

1 against their respective sealing faces by the leaf
2 springs. The resulting seal is actually two rotatable
3 seals placed in series.

4 One such sealing system is shown and described in
5 U.S. Patent No. 3,028,163 to Heinrich. Sealing systems
6 having only one rotatable seal are also known. All of
7 these sealing systems are complex and thus inherently
8 prone to failure.

SUMMARY OF THE INVENTION

The present invention is directed to a sealing system for a rotating machine having a stationary element and a drive element rotationally connected to the stationary element. The sealing system includes a plate having a bearing surface. The plate is rigidly connected to either the drive element or the stationary element. The sealing system also includes a sealing assembly having a resilient bellows secured to a thrust plate having a bearing surface. The bellows provides a force on the thrust plate forcing the bearing surface of the thrust plate against the bearing surface of the other plate to form a dynamic seal.

According to a further aspect of the invention, the resilient bellows comprises a collar to which the thrust plate is secured.

According to yet another aspect of the invention,
the sealing assembly has a static sealing element
disposed within a gap provided between the collar and the

1 thrust plate.

2 According to a even further aspect of the invention,
3 a mounting element provides the connection of the plate
4 to either the drive element or the stationary element.

5 BRIEF DESCRIPTION OF THE DRAWINGS

6 Fig. 1 shows a sectional side view of a portion of a
7 rotating machine having a seal according to the
8 invention; and

9 Fig. 2 shows a bellows seal assembly according to
10 the invention.

11 DESCRIPTION OF THE INVENTION

12 Referring to Fig. 1, a rotating machine 8, such as a
13 pump, includes a drive element or shaft 10, and a driven
14 element or work implement 12, such as an impeller,
15 propeller, or mixing bars. The implement 12 is secured
16 to the drive shaft 10 by a bolt 13. It should be
17 appreciated that while, in the present embodiment, the
18 drive element is a shaft, any suitable rotational drive
19 element could be substituted therefor.

20 The drive shaft 10 extends through a seal chamber 14
21 defined by a housing 16 of the machine 8. The housing 16
22 is a stationary element of the machine 8. A bellows
23 sealing assembly 17 is disposed on the drive shaft 10 in
24 the seal chamber 14. The bellows sealing assembly 17
25 comprises a resilient bellows 18 holding a thrust plate
26 20, 22 at each end.

1 A drive plate 24 is mounted to the drive shaft 10 by
2 a first mounting element or drive plate mount 26, such
3 that the drive plate rotates with the shaft 10. The
4 drive plate 24 has a bearing surface 28.

5 A bearing surface 30 at the free end of the first
6 thrust plate 20 bears on the bearing surface 28 of the
7 drive plate 24. As explained below in detail, a first
8 dynamic (rotatable) seal 32 is formed at the interface
9 between these bearing surfaces 28, 30.

10 A stationary plate 34 is mounted to the housing 16
11 by second mounting element or a stationary plate mount
12 36. The stationary plate 34 has a bearing surface 38.

13 A bearing surface 40 at the free end of the second
14 thrust plate 22 bears on the bearing surface 38 of the
15 stationary plate 34. As explained below in detail, a
16 second dynamic (rotatable) seal 42 is formed at the
17 interface between these bearing surfaces 38, 40.

18 It should be appreciated, that each of the
19 aforementioned plates 20, 22, 24, 34, could be
20 substituted with other structures having bearing surfaces
21 for sealing against. Further, in some applications, such
22 as machines that require minimal or infrequent rotation,
23 the mounts 26, 36 can be eliminated and the drive and
24 stationary plates 24, 34 can be mounted by other means,
25 for example by shrinking the plates 24, 34 into reverse
26 tapers or by gluing the plates into recessed holders.

27 A seal gland 44 is disposed over the stationary
28 plate mount 36. The seal gland 44 is secured to the

1 housing 16 of the seal chamber 14 by two bolts 45 and
2 thereby closes the seal chamber 14.

3 Referring Fig. 2, the bellows 18, as the term is
4 used herein, is a resilient tube which behaves like a
5 compression spring. In the embodiment of Fig. 2,
6 corrugations or ribs 46 provided to the bellows 18
7 provide a force 48 longitudinally along the axis of the
8 drive shaft 10. The bellows 18 imparts this force 48 on
9 the thrust plates 20, 22, biasing them outwardly against
10 their respective plates 24, 34.

11 The bellows 18 can be formed of thin wall metal or
12 plastic tubing, which can be seamless or welded. The
13 corrugations 46 are formed in the tubing by a known
14 hydraulic or rolling process. Alternatively, the bellows
15 18 can be formed by injection molding, by the lamination
16 of a tube to a coil spring, or by another suitable
17 process.

18 The bellows 18 also includes inwardly turned edges
19 at each end. The inwardly turned edges form somewhat
20 frustoconical or tapered collars 50, 52 for receiving the
21 respective thrust plates 20, 22 therein. The thrust
22 plates 20, 22 can be statically sealed to the bellows 18
23 by gaskets, such as elastomeric rings or sealants, such
24 as epoxy, disposed, for example, in respective gaps 54,
25 56, between the thrust plates 20, 22 and the collars 50,
26 52. With proper selection of the gaskets or sealants,
27 passage of molecules as small as nitrogen (N_2) can be
28 blocked.

1 During operation of the machine 8, rotation of the
2 drive shaft 10 causes rotation of the drive plate mount
3 26 and the drive plate 24. Depending on the instant
4 force of friction between the bearing surfaces 28, 30 of
5 the first dynamic seal 32 as compared to the instant
6 force of friction between the bearing surfaces 38, 40 of
7 the second dynamic seal 42, the bellows sealing assembly
8 17 will sometimes be driven by and rotate with the drive
9 shaft 10 and at other times remain stationary.

10 The thrust plates 20, 22 are made of carbon,
11 composite plastic, silicon carbide, or composite metal.
12 Each of the members 20, 22, 24, 34 that have bearing
13 surfaces 28, 30, 38, 40 are made of a graphite containing
14 material, such as graphite filled carbon or silicon
15 carbide. Relative rotation of bearing surfaces 28 and
16 30, 38 and 40 causes graphite in the members 20, 22, 24,
17 34 to form a lubricating graphite film therebetween. In
18 this way, the respective interfaces between the drive
19 plate 24, the stationary plate 34, and the thrust plates
20 20, 22 provide the dynamic seals 32, 42.

21 In the embodiment shown in FIGS. 1 and 2, the
22 sealing assembly 17 provides two dynamic seals 32, 42 and
23 the sealing assembly 17 alternates between driven and
24 stationary operation (as explained above). According to
25 alternative embodiments, a single seal can be provided.

26 A single seal can be provided, as one alternative,
27 by securing one end of the bellows 18 directly to the
28 stationary mount 36 so that the bellows 18 becomes a

1 permanently stationary element. In this alternative, the
2 second thrust plate 22 and the stationary plate 34 are
3 eliminated.

4 A single seal could also be provided, as another
5 alternative, by securing one end of the bellows 18
6 directly to the drive shaft 10 so that the bellows
7 becomes a driven element. In this alternative, the first
8 thrust plate 20 and the drive plate 24 are eliminated.

9 The present disclosure describes several embodiments
10 of the invention, however, the invention is not limited
11 to these embodiments. Other variations are contemplated
12 to be within the spirit and scope of the invention.